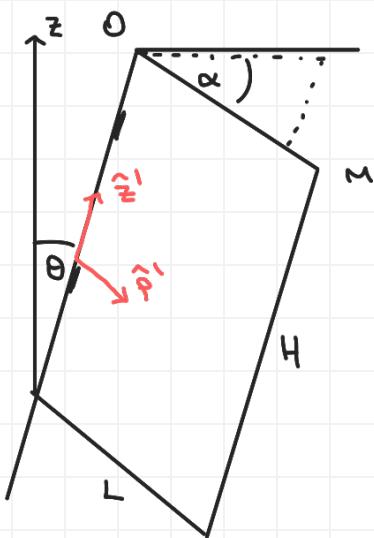


P1



$$E = \frac{1}{2} I \omega^2 + U$$

I: mom. de inercia de la puerta c/r al eje O

momento de inercia:

$$I = \int r^2 dm \quad r: \text{distancia al eje } (r^2 = p^2)$$

$$dm = \sigma dS \quad \text{pero} \quad \sigma = \frac{M}{LH}$$

$$dm = \frac{M}{LH} dp dz$$

$$\begin{aligned} I &= \iint_{-H/2}^{H/2} \iint_0^L p^2 \cdot \frac{M}{LH} dp dz \\ &= \frac{M}{LH} \int_{-H/2}^{H/2} dz \cdot \int_0^L p^2 dp \\ &= \frac{M}{LH} \cdot H \cdot \frac{1}{3} L^3 \end{aligned}$$

$$I = \frac{1}{3} ML^2$$

centro de masas:

- puerta homogénea \rightarrow c.m. en el centro de la puerta

$$\vec{r}_{cm} = \frac{L}{2} \hat{p}'$$

energía potencial:

$$U = Mgz$$

$$z = r' \sin \theta \sin \psi + \phi' \sin \theta \cos \psi + z' \cos \theta$$

$$r' = \frac{L}{2} \quad \phi' = 0 \quad z' = 0$$

$$U_{\text{cm}} = Mg \cdot \frac{L}{2} \sin \theta \sin \psi$$

$$E_{\text{cerrada}} = E_{\text{abierta}}(\psi, \dot{\psi})$$

$$0 = \frac{1}{2} \cdot \frac{1}{3} M L^2 \cdot \dot{\psi}^2 + \frac{M g L}{2} \sin \theta \sin \psi$$

$$0 = \frac{1}{3} L \dot{\psi}^2 + g \sin \theta \sin \psi$$

$$\dot{\psi} = \sqrt{-\frac{3g}{L} \sin \theta \sin \psi}$$

resolución:

$$\text{pos. inicial: } \psi = -\alpha = -\pi/2$$

$$\text{pos. final: } \psi = 0$$

$$\frac{d\psi}{dt} = \sqrt{-\frac{3g}{L} \sin \theta \sin \psi}$$

$$\int_0^T dt = \int_{-\pi/2}^0 \left(-\frac{3g}{L} \sin \theta \sin \psi \right)^{-1/2} d\psi$$

$$T = \left(\frac{3g}{L} \sin \theta \right)^{-1/2} \underbrace{\int_{-\pi/2}^0 \frac{1}{\sqrt{1-\sin \psi}} d\psi}_{J=2,62}$$

$$T = \sqrt{\frac{L}{3g \sin \theta}} \cdot J$$

$$\sqrt{\sin \theta} = \frac{\pi}{T} \sqrt{\frac{L}{3g}}$$

$$\theta = \arcsin\left(\frac{\pi^2}{T^2} \cdot \frac{L}{3g}\right)$$

$$\theta = \arcsin\left(\frac{2,62^2}{2^2} \cdot \frac{1}{30}\right)$$

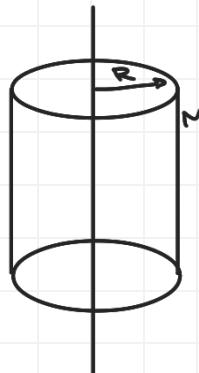
$$= \arcsin(0,057)$$

$$\theta = 3,279^\circ$$

P2

- cilindro hueco masa M_1 radio R_1 r.s.r. en la superficie interna de otro cilindro hueco de masa M_2 y radio R_2 ($R_1 \ll R_2$)
- cilindros con ejes paralelos; 2 es libre de rotar.
- calc. freq. peq. osc.

mom. inercia cascarón cilíndrico



$$\begin{aligned}
 I &= \int r^2 dm \\
 &= R^2 \int \sigma dS \\
 &= R^2 \cdot \frac{M}{2\pi RH} \cdot \iint_0^{2\pi} R d\phi dz \\
 &= \frac{MR^2}{2\pi H} \cdot \int_0^H dz \cdot \int_0^{2\pi} d\phi
 \end{aligned}$$

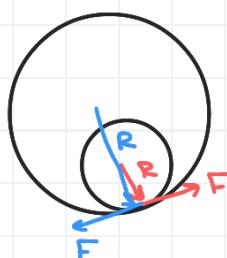
$$I = MR^2$$

$$I_1 = M_1 R_1^2$$

$$I_2 = M_2 R_2^2$$

ecs. de fuerzas:

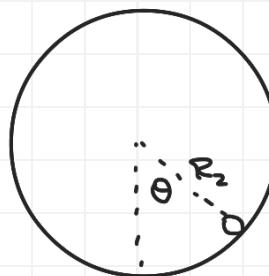
r.s.r $\rightarrow F$ entre cilindros



$$I_1 \alpha_1 = R_1 F$$

$$I_2 \alpha_2 = -R_2 F$$

$$R_1 \ll R_2 \quad \longrightarrow$$



Luego, la condición de no deslizamiento entre cilindros:

$$R_2 \theta_2 - R_1 \theta_1 \approx R_2 \theta \quad / \frac{d^2}{dt^2}$$

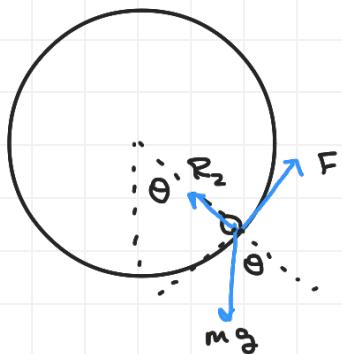
$$M_1 R_1^2 \ddot{\theta}_1 = R_1 F \rightarrow R_1 \ddot{\theta}_1 = \frac{F}{M_1}$$

$$-M_2 R_2^2 \ddot{\theta}_2 = R_2 F \rightarrow -R_2 \ddot{\theta}_2 = \frac{F}{M_2}$$

$$R_1 \ddot{\theta}_1 - R_2 \ddot{\theta}_2 = F \left(\frac{1}{M_1} + \frac{1}{M_2} \right)$$

$$-R_2 \ddot{\theta} = F \left(\frac{1}{M_1} + \frac{1}{M_2} \right)$$

fuerzas sobre M₁



tangenciales: F y $-M_1 g \sin\theta$

$$M_1 (R_2 \ddot{\theta}) = F - M_1 g \sin\theta$$

$$-R_2 \ddot{\theta} = (M_1 R_2 \ddot{\theta} + M_1 g \sin\theta) \left(\frac{1}{M_1} + \frac{1}{M_2} \right)$$

$$-R_2 \ddot{\theta} = R_2 \ddot{\theta} + g \sin\theta + \frac{M_1}{M_2} R_2 \ddot{\theta} + \frac{M_1}{M_2} g \sin\theta$$

$$\ddot{\theta} = \ddot{\theta} \left(2 + \frac{M_1}{M_2} \right) R_2 + \left(1 + \frac{M_1}{M_2} \right) g \sin\theta$$

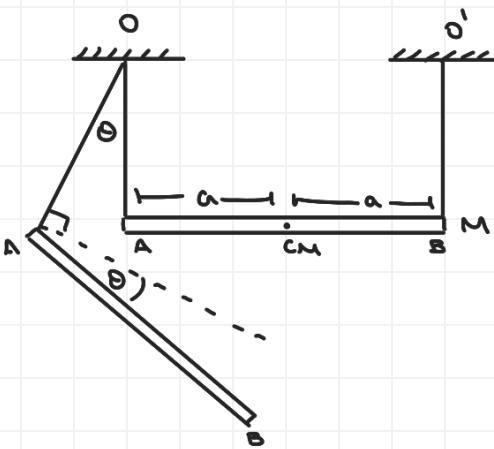
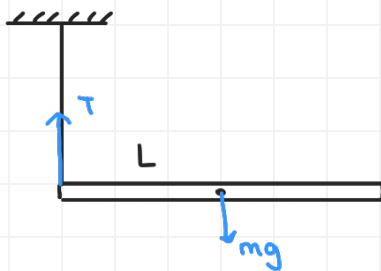
$$\ddot{\theta} = \ddot{\theta} (2M_2 + M_1) R_2 + (M_1 + M_2) g \sin\theta$$

$$\ddot{\theta} = \ddot{\theta} + \frac{M_1 + M_2}{M_1 + 2M_2} \cdot \frac{g}{R_2} \sin\theta$$

P.O. $\sin\theta \approx \theta$

$$\omega = \sqrt{\frac{M_1 + M_2}{M_1 + 2M_2}} \sqrt{\frac{g}{R_2}}$$

P3

Corte:

$$ma = T - mg$$

$$I\alpha = -Lmg$$

$$L\alpha = a$$

$$mL\alpha = T - mg$$

$$mL \cdot -\frac{Lmg}{I} = T - mg$$

$$-\frac{m^2 L^2 g}{I} + mg = T$$

$$\begin{aligned} I &= \int r^2 dm = \lambda \int_0^{z_L} r^2 dr \\ &= \frac{M}{2L} \cdot \frac{8L^3}{3} = \frac{4}{3} ML^2 \end{aligned}$$

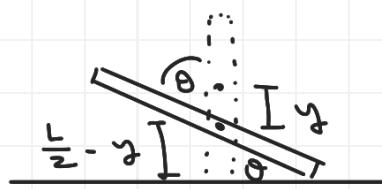
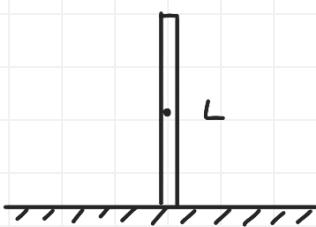
$$-\frac{M^2 L^2 g}{\frac{4}{3} ML^2} + Mg = T$$

$$-\frac{3}{4} Mg + Mg = T$$

$$\boxed{\frac{1}{4} Mg = T}$$

P4

- vara largo L masa M dist. uniforme
- inicialmente pos. vertical en mesa sin roce
- calcular rapidez del c.m. como fn. del ángulo de caída.



Como no hay fuerzas horizontales, el C.M. cae directamente hacia abajo.

Buscamos $v(\theta) \rightarrow$ cons. de energía.

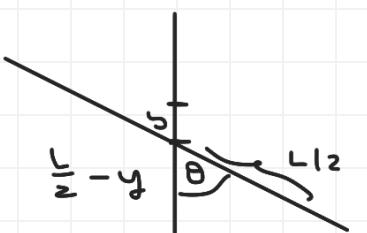
$$E_i = 0 + Mg z_{cm} = Mg \frac{L}{2}$$

$$K_f = \frac{1}{2} I \dot{\theta}^2 + \frac{1}{2} M \dot{y}^2$$

$$U_f = Mg \left(\frac{L}{2} - y \right)$$

$$\frac{1}{2} I \dot{\theta}^2 + \frac{1}{2} M \dot{y}^2 + Mg \cancel{\frac{L}{2}} - Mg y \cancel{\frac{L}{2}} = Mg \cancel{\frac{L}{2}}$$

$$I \dot{\theta}^2 + M \dot{y}^2 = 2Mgy$$



$$\cos \theta = \frac{L/2 - y}{L/2}$$

$$\frac{L}{2} \cos \theta = \frac{L}{2} - y$$

$$y = \frac{L}{2} (1 - \cos \theta)$$

$$\dot{y} = \frac{L}{2} \dot{\theta} \sin \theta$$

$$\frac{2\dot{y}}{L \sin \theta} = \dot{\theta}$$

$$I \cdot \frac{4\dot{\gamma}^2}{L^2 \sin^2 \theta} + Mg^2 = 2Mg \frac{L}{2} (1 - \cos \theta)$$

$$\left(\frac{4I}{L^2 \sin^2 \theta} + M \right) \dot{\gamma}^2 = MgL(1 - \cos \theta)$$

$$\begin{aligned} I &= \int_{-L/2}^{L/2} r^2 dm \\ &= \lambda \int_{-L/2}^{L/2} y^2 dy \quad \lambda = \frac{M}{L} \\ &= \frac{M}{L} \cdot \frac{1}{3} \left[\frac{L^3}{8} + \frac{L^3}{8} \right] \\ &= \frac{1}{12} ML^2 \end{aligned}$$

$$\left(\frac{4}{L^2 \sin^2 \theta} \cdot \frac{1}{12} ML^2 + M \right) \dot{\gamma}^2 = MgL(1 - \cos \theta)$$

$$\boxed{\left(\frac{1}{3 \sin^2 \theta} + 1 \right) \dot{\gamma}^2 = gL(1 - \cos \theta)}$$

