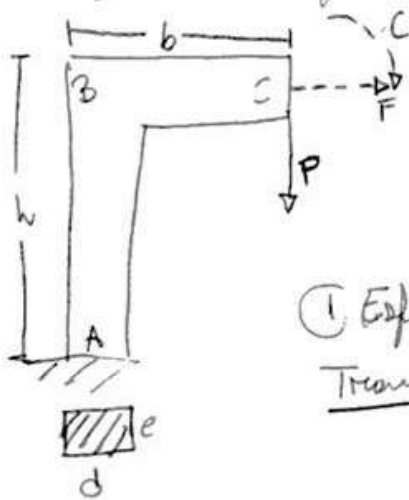


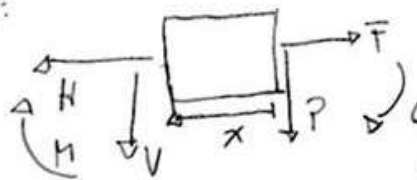
Problema P1) Lix Artiglas, $\delta_y, \delta_x, \theta_z$?



Como se debe conocer la deflexión en x y el ángulo de rotación, se agregan la fuerza y momento ficticio $\vec{F}$ y $\vec{C}$ respectivamente. Aplicados en el punto C. $F_y C$ a analizar el aplica Castigliano

1) Esfuerzos internos.

Tramo BC:



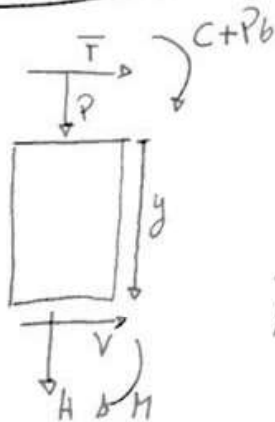
$\Rightarrow E_c$ de equilibrio

$$\sum F_x: H = F \rightarrow \text{tracción}$$

$$\sum F_y: V = -P \rightarrow \text{corte}$$

$$\sum M_z: M = -C - Px \rightarrow \text{flexión}$$

Tramo AB:



Se suma el torque generado por P

$$\sum F_x: V = -F \rightarrow \text{corte}$$

$$\sum F_y: H = -P \rightarrow \text{compresión}$$

$$\sum M_z: M = -C - Pb - Fy \rightarrow \text{flexión}$$

2) Energía elástica: cada fuerza

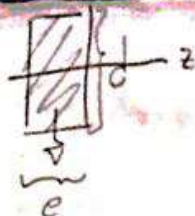
Tracción/Compresión: $U_{tr} = \frac{P^2 L}{2EA} \Rightarrow U_{tr} = U_{tr}^{AB} + U_{tr}^{BC} = \frac{P^2 h}{2Eed} + \frac{F^2 b}{2Eed} = \frac{1}{2Eed} (P^2 h + F^2 b)$

Corte $U_{co} = \frac{3}{5} \int_0^h \frac{V^2(x) dx}{GA} \rightarrow U_{co} = U_{co}^{AB} + U_{co}^{BC} = \frac{3}{5} \int_0^h \frac{F^2 dy}{Ged} + \frac{3}{5} \int_0^b \frac{P^2 dx}{Ged}$

$$= \frac{3}{5Ged} (F^2 h + P^2 b)$$

Flexion: $U_{fl} = \int_0^L \frac{M^2(x)}{2EI_z} dx$

$I_z = \frac{ed^3}{12}$



$= U_{fl}^{AB} + U_{fl}^{BC}$

$U_{fl}^{AB} = \int_0^h \frac{(-c - Pb - Fy)^2}{2EI_z} dy = \frac{6}{Eed^3} \int_0^h (c + Pb + Fy)^2 dy$

C.V. $\rightarrow u = c + Pb + Fy$
 $du = Fdy$

$= \frac{6}{Eed^3} \int_{c+Pb}^{c+Pb+FH} \frac{u^2}{F} du = \frac{6}{FEed^3} \left(\frac{(c+Pb+FH)^3}{3} - \frac{(c+Pb)^3}{3} \right)$

$= \frac{2}{FEed^3} \left(F^3h^3 + 3F^2h^2(c+Pb) + 3Fh(c+Pb)^2 + (c+Pb)^3 - (c+Pb)^3 \right)$

$= \frac{2}{FEed^3} \left(F^3h^3 + 3F^2h^2(c+Pb) + 3Fh(c+Pb)^2 \right)$

$= \frac{2(F^3h^3 + 3F^2h^2(c+Pb) + 3Fh(c+Pb)^2)}{Eed^3}$

$U_{fl}^{BC} = \int_0^b \frac{(-c - Px)^2}{2EI_z} dx = \frac{2(P^2b^3 + 3Pb^2c + 3bc^2)}{Eed^3}$

$U_{fl} = \frac{2}{Eed^3} (F^3h^3 + 3F^2h^2(c+Pb) + 3Fh(c+Pb)^2) + \frac{2}{Eed^3} (P^2b^3 + 3Pb^2c + 3bc^2)$

$= \frac{2}{Eed^3} \left(F^3h^3 + P^2b^3 + 3F^2h^2(c+Pb) + 3Pb^2c + 3h(c+Pb)^2 + 3bc^2 \right)$

$U_T = U_{cor}^{AB} + U_{fl}^{AB} + U_{cor}^{BC} + U_{fl}^{BC} + U_{fl}^{AB} + U_{fl}^{BC}$

$= \frac{Ph}{2Ecd} + \frac{6Pb}{56ed} + \frac{2}{Eed^3} \left(F^3h^3 + P^2b^3 + 3F^2h^2(c+Pb) + 3Pb^2c + 3h(c+Pb)^2 + 3bc^2 \right)$

③ Castigliano: $\delta_y = \frac{\partial U_T}{\partial P} \Big|_{c=F=0}$ $\delta_x = \frac{\partial U_T}{\partial F} \Big|_{c=F=0}$ $\theta_c = \frac{\partial U_T}{\partial c} \Big|_{c=F=0}$

$\frac{\partial U_T}{\partial P} = \frac{Ph}{Eed} + \frac{6Pb}{56ed} + \frac{2}{Eed^3} \left(2Pb^3 + 3F^2h^2b + 3b^2c + 6h(c+Pb)b \right) = \delta_y$

$\Rightarrow \frac{\partial U_T}{\partial P} \Big|_{c=F=0} = \frac{Ph}{Eed} + \frac{6Pb}{56ed} + \frac{2}{Eed^3} \left(2Pb^3 + 6Phb^2 \right) = \frac{4P}{Eed} \left(\frac{h}{4} + \frac{b^3 + 3b^2h}{d^2} \right) + \frac{6Pb}{56ed} = \delta_y$

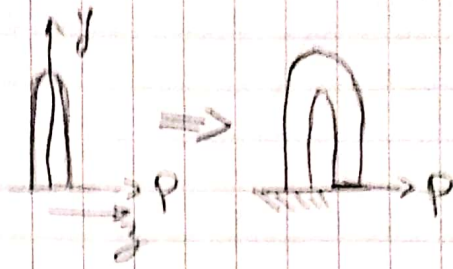
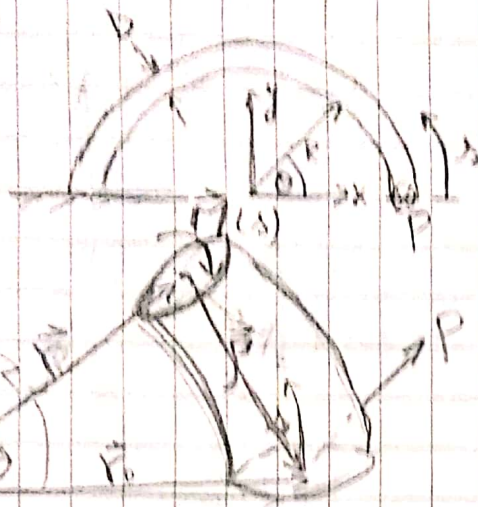
$$\delta_x = \frac{\partial U}{\partial F} \bigg|_{\substack{C=0 \\ F=0}} = \frac{Fb}{Eed} + \frac{6Fh}{5Ged} + \frac{2}{Eed^3} (2Fh^3 + 3h^2(C+Pb))$$

$$= \frac{6Pbh^2}{Eed^3}$$

$$\theta_c = \frac{\partial U}{\partial C} \bigg|_{C=F=0} = \frac{2}{Eed^3} (3Fh^2 + 3Pb^2 + 6h(C+Pb) + 6bC)$$

$$= \frac{2}{Eed^3} (3Pb^2 + 6Pbh) = \frac{6Pb}{Eed^3} (b + 2h) //$$

P211



* Solo consideraremos flexión y torsión.

$$R = R + \frac{P}{2} = 0,48 \text{ [m]}$$

$$\vec{r} = R \cos(\theta) \hat{i} + R \sin(\theta) \hat{j} = R \cos\left(\frac{\Delta}{R}\right) \hat{i} + R \sin\left(\frac{\Delta}{R}\right) \hat{j}$$

$$\vec{M}(\Delta) = \vec{r} \times \vec{P}$$

$$\vec{P} = -P \hat{k}$$

$$\vec{r} = \vec{r}_0 - \vec{r} \quad \text{con } \vec{r}_0 = R \hat{i}$$

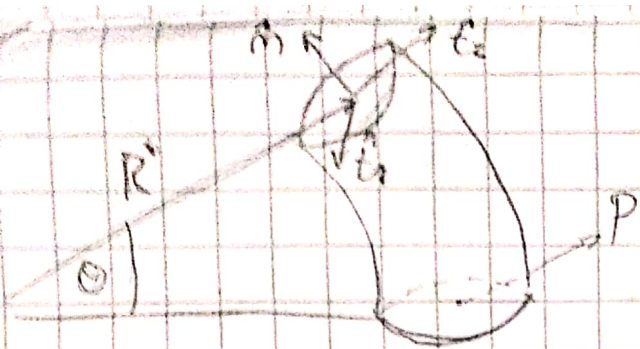
$$\Rightarrow \vec{r} = R \hat{i} - R \cos\left(\frac{\Delta}{R}\right) \hat{i} - R \sin\left(\frac{\Delta}{R}\right) \hat{j}$$

$$\Rightarrow \vec{r} = R \left(1 - \cos\left(\frac{\Delta}{R}\right)\right) \hat{i} - R \sin\left(\frac{\Delta}{R}\right) \hat{j}$$

$$\sum M = 0$$

$$\Rightarrow \vec{M} = - \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ R(1 - \cos(\frac{\Delta}{R})) & -R \sin(\frac{\Delta}{R}) & 0 \\ 0 & 0 & -P \end{vmatrix} = -P \cdot R \cdot \sin\left(\frac{\Delta}{R}\right) \hat{i} - P \cdot R \cdot \left[1 - \cos\left(\frac{\Delta}{R}\right)\right] \hat{j}$$

→ Tenemos el momento parametrizado, ahora hay que saber que componente genera flexión y torsión
 → Para esto, hay que tener un sistema coordenado en la cara donde se mide el momento.



* El producto $\vec{M} \cdot \hat{n}$ generará torsión
 los productos $\vec{M} \cdot \hat{t}_1$ y $\vec{M} \cdot \hat{t}_2$ generarán flexión

$$\hat{n} = \frac{\frac{d\vec{r}}{d\lambda}}{\left\| \frac{d\vec{r}}{d\lambda} \right\|} = \frac{-\sin\left(\frac{\lambda}{R}\right)\hat{x} + \cos\left(\frac{\lambda}{R}\right)\hat{y}}{1} = -\sin\left(\frac{\lambda}{R}\right)\hat{x} + \cos\left(\frac{\lambda}{R}\right)\hat{y}$$

$\hat{t}_1$ es tal que $\hat{t}_1 \cdot \hat{n} = 0$

$$\begin{pmatrix} t_{11} \\ t_{12} \\ t_{13} \end{pmatrix} \cdot \begin{pmatrix} -\sin\left(\frac{\lambda}{R}\right) \\ \cos\left(\frac{\lambda}{R}\right) \\ 0 \end{pmatrix} = -t_{11} \cdot \sin\left(\frac{\lambda}{R}\right) + t_{12} \cdot \cos\left(\frac{\lambda}{R}\right) = 0$$

$\Rightarrow$ puede ser $t_{11} = \cos\left(\frac{\lambda}{R}\right)$ y $t_{12} = \sin\left(\frac{\lambda}{R}\right)$

$$\Rightarrow \hat{t}_1 = \hat{t}_1 = \cos\left(\frac{\lambda}{R}\right)\hat{x} + \sin\left(\frac{\lambda}{R}\right)\hat{y}$$

$$\hat{t}_2 = \hat{n} \times \hat{t}_1 = \begin{bmatrix} 0 & 0 & 1 \end{bmatrix} = \hat{z}$$

$$\therefore M_T = \vec{M} \cdot \hat{n} = -P \cdot R \left[\cos\left(\frac{\lambda}{R}\right) - 1 \right] = T(\lambda)$$

$$\textcircled{2} M_{F1} = \vec{M} \cdot \hat{t}_1 = -P \cdot R \cdot \sin\left(\frac{\lambda}{R}\right) = M(\lambda)$$

$$\textcircled{3} M_{F2} = \vec{M} \cdot \hat{t}_2 = 0$$

$$U_{\text{torsion}} = \int_0^{\pi R} \frac{M_T^2(\lambda)}{2GJ} d\lambda = \frac{3\pi P^2 R^3}{4GJ}$$

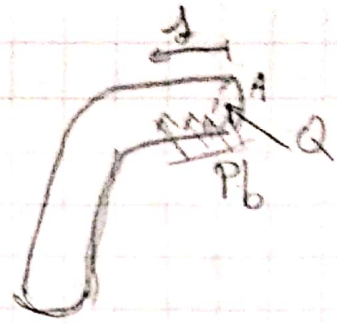
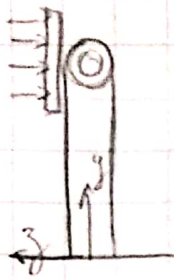
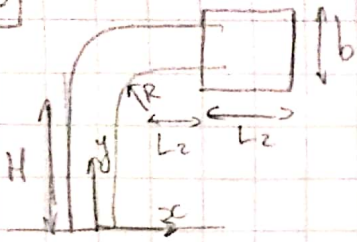
$$U_{\text{flexion}} = \int_0^{\pi R} \frac{M_{F1}^2(\lambda)}{2EI_z} d\lambda = \frac{\pi P^2 R^3}{4EI_z}$$

$$\begin{aligned} J &= \frac{\pi R^4}{2} = 2,5 \cdot 10^{-7} \\ I_z &= \frac{\pi R^4}{4} = 1,3 \cdot 10^{-7} \\ \Rightarrow P &= \frac{2\delta_p}{\pi R^3} \left(\frac{1}{GJ} + \frac{1}{EI_z} \right)^{-1} = 544 \text{ N} \end{aligned}$$

$$\Rightarrow U_{\text{tot}} = \frac{\pi P^2 R^3}{4} \left(\frac{3}{GJ} + \frac{1}{EI_z} \right)$$

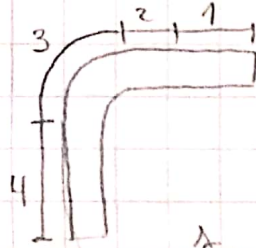
$$\Rightarrow \delta_p = \frac{dU_{\text{tot}}}{dP} = \frac{\pi P R^3}{2} \left(\frac{3}{GJ} + \frac{1}{EI_z} \right)$$

P3



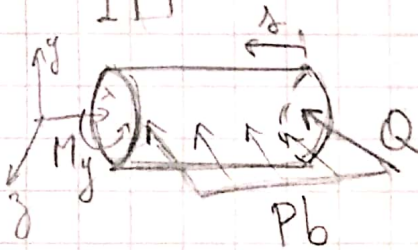
* Se considerará energía por flexión y torsión

4 tramos:



Tramo 1

$$0 < s < L_2$$

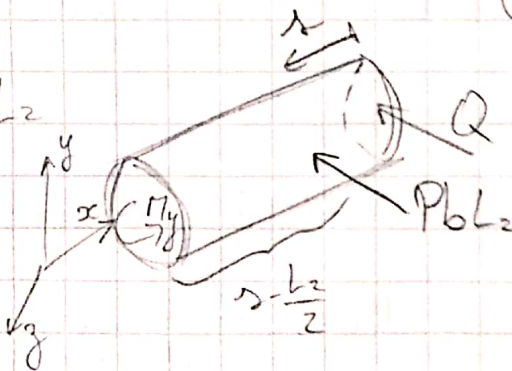


$$\sum M = M_y + \frac{P_b \cdot s^2}{2} + Q \cdot s = 0$$

$$M_y(s) = -Qs - \frac{P_b s^2}{2}$$

Tramo 2

$$L_2 < s < L_1 + L_2$$

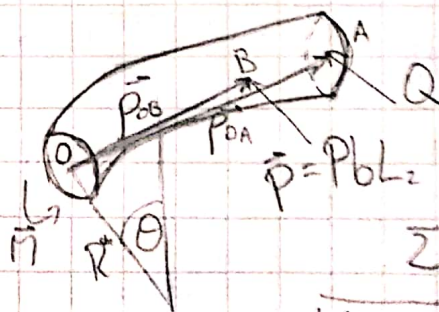


$$M_y(s) = -Qs - P_b L_2 \left(s - \frac{L_2}{2} \right)$$

Tramo 3

$$0 < \theta < \frac{\pi}{2}$$

$$R^* = R + \frac{P}{2}$$



$$\vec{Q} = -Q \hat{x}$$

$$\vec{P} = -P_b L_2 \hat{x}$$

$$\sum M_0: \vec{M} + \vec{r}_{0B} \times \vec{P} + \vec{r}_{0A} \times \vec{Q} = 0$$

con:

$$\vec{r}_{0A} = [R^* \sin(\theta) + L_1 + L_2] \hat{x} + (1 - \cos(\theta)) \cdot R^* \hat{y}$$

$$\Rightarrow \vec{M} = -\vec{r}_{0A} \times \vec{Q} - \vec{r}_{0B} \times \vec{P}$$

$$\vec{r}_{0B} = [R^* \sin(\theta) + L_1 + \frac{L_2}{2}] \hat{x} + (1 - \cos(\theta)) R^* \hat{y}$$

$$-\vec{p}_{od} \times \vec{Q} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ R^* \sin(\theta) + L_1 + L_2 & R^* (1 - \cos(\theta)) & 0 \\ 0 & 0 & -Q \end{vmatrix}$$

$$= R^* \cdot Q (1 - \cos(\theta)) \hat{i} - Q (R^* \sin(\theta) + L_1 + L_2) \hat{j}$$

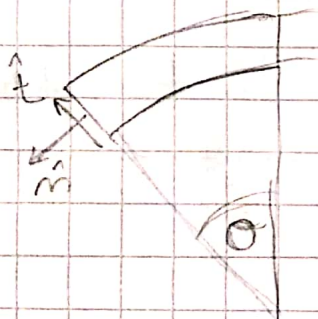
$$-\vec{p}_{ob} \times \vec{P} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ R^* \sin(\theta) + L_1 + \frac{L_2}{2} & R^* (1 - \cos(\theta)) & 0 \\ 0 & 0 & -P_b L_2 \end{vmatrix}$$

$$= R^* (Q + P_b L_2) (1 - \cos(\theta)) \hat{i} - (2Q + P_b L_2) (R^* \sin(\theta) + L_1 + \frac{L_2}{2}) \hat{j}$$

$$\Rightarrow \vec{M}(\theta) = R^* \left[Q (1 - \cos(\theta)) + (Q + P_b L_2) (1 - \cos(\theta)) \right] \hat{i}$$

$$- \left[Q (R^* \sin(\theta) + L_1 + L_2) + (2Q + P_b L_2) (R^* \sin(\theta) + L_1 + \frac{L_2}{2}) \right] \hat{j}$$

→ ahora hay que determinar las componentes de torsión y flexión.



$$\hat{m} = -\cos(\theta) \hat{i} - \sin(\theta) \hat{j}$$

$$\hat{t} = \cos(\theta) \hat{j} - \sin(\theta) \hat{i}$$

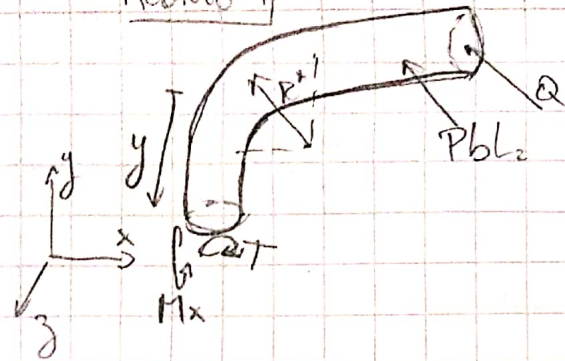
$$M_{\text{trans}} = \vec{M} \cdot \hat{t} = M(\theta) \rightarrow \text{componente que causa flexión}$$

$$M_{\text{norm}} = \vec{M} \cdot \hat{m} = T(\theta) \rightarrow \text{componente que causa torsión.}$$

$$\Rightarrow M(\theta) = M_{\text{trans}} = - (1 - \cos(\theta)) \sin(\theta) \cdot R^* (Q + P_b L_2) - \left[R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right] \cos(\theta) \cdot (2Q + P_b L_2)$$

$$T(\theta) = M_{\text{norm}} = - (1 - \cos(\theta)) \cdot \cos(\theta) \cdot R^* (Q + P_b L_2) + \left[R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right] \sin(\theta) (2Q + P_b L_2)$$

Tirano 4/



$$\sum M_y: (R^* + L_1 + \frac{L_2}{2}) P b L_2 + (R^* + L_1 + L_2) Q + T = 0$$

$$\Rightarrow T(y) = (R^* + L_1 + \frac{L_2}{2}) P b L_2 - (R^* + L_1 + L_2) \cdot Q$$

$$\sum M_x: M_x - (R^* + y)(P b L_2 + Q) = 0$$

$$\Rightarrow M_x(y) = (R^* + y)(P b L_2 + Q)$$

Con todas las expresiones se puede calcular la energía total.

$$U_{\text{tot}} = \int_0^{L_2} \frac{M_y^2(\lambda)}{2EI_z} d\lambda + \int_{L_2}^{L_1+L_2} \frac{M_y^2(\lambda)}{2EI_z} d\lambda + \int_0^{\pi/2} \frac{M_z^2(\theta)}{2EI_z} \cdot R^* d\theta + \int_0^{\pi/2} \frac{I^2(\theta)}{2GJ} R^* d\theta$$

$$+ \int_0^H \frac{M_x^2(y)}{2EI_z} dy + \int_0^H \frac{T^2(y)}{2GJ} dy \rightarrow \text{se puede hacer con TI o a mano, aquí lo haremos a mano}$$

Usando el teorema de castiglione:

$$\delta_{\lambda_2} = \frac{\partial U_{\text{tot}}}{\partial Q} \Big|_{Q=0}$$

* Para resolver la integral usaremos el hecho de que:

$$\frac{dF^2(x)}{dQ} = 2F(x) \frac{dF(x)}{dQ} \quad (\text{Regla de la cadena})$$

entonces nos interesa:

$$\bullet 0 < \lambda < L_2: \frac{dM_y}{dQ} = -x$$

$$\bullet L_2 < \lambda < L_1 + L_2: \frac{dM_y}{dQ} = -x$$

• $0 < \theta < \frac{\pi}{2}$: $\frac{dM}{dQ} = -(1 - \cos(\theta)) \sin(\theta) R^* - 2 \left[R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right] \cdot \cos(\theta)$

$\frac{dT}{dQ} = -(1 - \cos(\theta)) \cos(\theta) R^* + 2 \left[R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right] \sin(\theta)$

• $0 < y < H$: $\frac{dM_x}{dQ} = R^* + y$ $\frac{dT}{dQ} = (R^* + L_1 + L_2)$

$$\begin{aligned} \delta A_B &= \frac{1}{EI_B} \left\{ \int_0^{L_2} M_y(s) \frac{\partial M_y(s)}{\partial Q} ds + \int_{L_2}^{L_1+L_2} M_y(s) \frac{\partial M_y}{\partial Q} ds + \int_0^{\pi/2} M(\theta) \frac{\partial M}{\partial Q} R^* d\theta \right. \\ &\quad \left. + \int_0^H M_x(y) \frac{\partial M_x(y)}{\partial Q} dy \right\} + \frac{1}{GJ} \left\{ \int_0^{\pi/2} T(\theta) \frac{\partial T}{\partial Q} R^* d\theta + \int_0^H T(y) \frac{\partial T}{\partial Q} dy \right\} \\ &= \frac{1}{EI_B} \left\{ \int_0^{L_2} (Qs + Pb s^2) s ds + \int_{L_2}^{L_1+L_2} [Qs + Pb L_2 (s - \frac{L_2}{2})] s ds \right. \\ &\quad + \int_0^{\pi/2} \left\{ (1 - \cos(\theta)) \sin(\theta) R^* (Q + Pb L_2) + \left(R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right) \cos(\theta) \cdot (2Q + Pb L_2) \right. \\ &\quad \left. + \left(R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right) \cos(\theta) (2Q + Pb L_2) \right\} \left\{ (1 - \cos(\theta)) \sin(\theta) R^* \right. \\ &\quad \left. + 2 \left(R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right) \cos(\theta) \right\} R^* d\theta + \int_0^H (R^* + y) (Pb L_2 + Q) (R^* + y) dy \Big\} \\ &\quad + \frac{1}{GJ} \left\{ \int_0^{\pi/2} \left[-(1 - \cos(\theta)) \cos(\theta) R^* \cdot (Q + Pb L_2) + \left(R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right) \sin(\theta) \right. \right. \\ &\quad \left. \cdot (2Q + Pb L_2) \right] \cdot \left[-(1 - \cos(\theta)) \cos(\theta) R^* + 2 \left(R^* \sin(\theta) + L_1 + \frac{L_2}{2} \right) \sin(\theta) \right] R^* d\theta \\ &\quad \left. + \int_0^H \left[\left(R^* + L_1 + \frac{L_2}{2} \right) Pb L_2 + \left(R^* + L_1 + L_2 \right) Q \right] (R^* + L_1 + L_2) dy \right\} \end{aligned}$$

Además

$$I = \frac{\pi}{64} [D^4 - (D-2e)^4] = 8,0576 \cdot 10^{-6} \text{ [m}^4\text{]}$$

$$J = \frac{\pi}{32} [D^4 - (D-2e)^4] = 2I$$

$$\Rightarrow \delta_{Az} = 0,00612 + 0,000052 \cdot Q = 0$$

$$\Rightarrow \boxed{\delta_{Az} = 6,12 \text{ [mm]}}$$