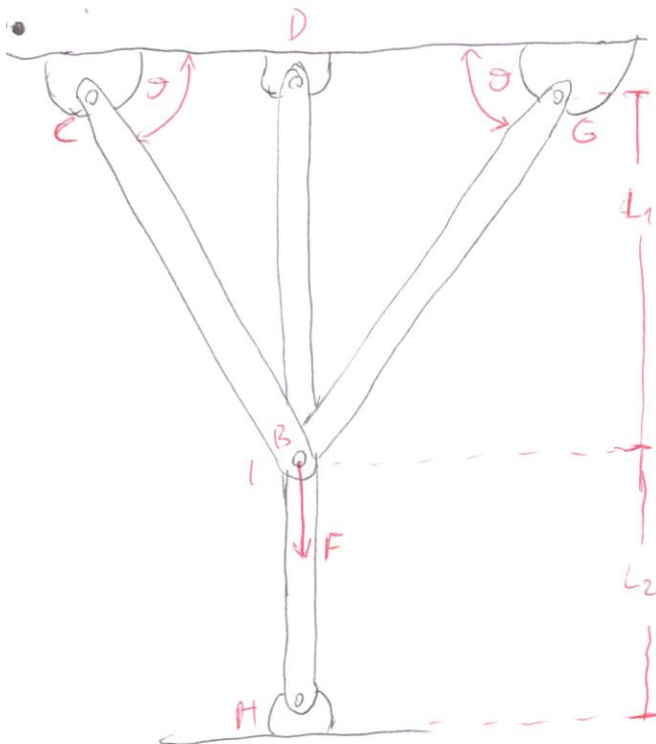
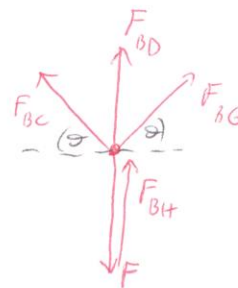




* DCL pasador B



Simetría

$$F_{BG} = F_{BC}$$

$$\sum F_y = 0 \Rightarrow F + F_{BH} - F_{BD} - 2F_{BG} \sin \theta = 0$$

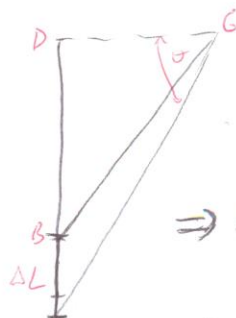
$$\boxed{\therefore F = F_{BH} + F_{BD} + 2F_{BG} \sin \theta} \quad (*)$$

* Aplicando Ley de Hooke sobre cada barra:

$$\sigma_{BH} = \frac{F_{BH}}{A} = E \cdot \epsilon_{BH} = E \frac{\Delta L}{L_{BH}} \leadsto \boxed{F_{BH} = AE \frac{\Delta L}{L_{BH}}} \quad (ii)$$

$$\sigma_{BD} = \frac{F_{BD}}{A} = E \cdot \epsilon_{BD} = E \frac{\Delta L}{L_{BD}} \leadsto \boxed{F_{BD} = AE \frac{\Delta L}{L_{BD}}} \quad (iii)$$

* ¿Qué pasa con BG?



Suponemos pequeñas deformaciones
 $\hookrightarrow \theta$ constante

$$\Rightarrow L_{BG}(\text{final}) = \frac{L_{BD} + \Delta L}{\sin \theta} \quad \wedge L_{BG} = \frac{L_{BD}}{\sin \theta}$$

$$\Rightarrow \Delta L_{BG} = L_{BG}(\text{final}) - L_{BG} = \frac{L_{BD} + \Delta L}{\sin \theta} - \frac{L_{BD}}{\sin \theta} \quad \left. \vphantom{\Delta L_{BG}} \right\} \boxed{\Delta L_{BG} = \frac{\Delta L}{\sin \theta}}$$

$$\therefore \sigma_{BG} = \frac{F_{BG}}{A} = E \cdot \epsilon_{BG} = E \cdot \frac{\Delta L}{L_{BG} \sin \theta} \leadsto \boxed{F_{BG} = A \cdot E \cdot \frac{\Delta L}{L_{BG} \sin \theta}} \quad (**)$$

$$\therefore F_{B6} = A \cdot E \cdot \frac{\Delta L}{L_{BD}} \quad (\text{in})$$

* Reemplazando en $\star$:

$$F = A E \frac{\Delta L}{L_{BH}} + A E \frac{\Delta L}{L_{BD}} + 2 A E \frac{\Delta L}{L_{BD}} \sin \theta \quad \begin{array}{l} L_{BH} = L_2 \\ L_{BD} = L_1 \end{array}$$

$$\therefore \Delta L = \frac{F}{A E \left(\frac{1}{L_2} + \frac{1}{L_1} + \frac{2 \sin \theta}{L_1} \right)} = 5,032 \cdot 10^{-5} \text{ m}$$

* Def. factor de Seguridad $\leadsto$ F.S. = $\frac{\sigma_c}{\sigma_{adm}} \Rightarrow \sigma_{adm} = \frac{\sigma_c}{F.S.} = \frac{200}{1,5}$

$\hookrightarrow$ Hay falla si $\sigma > \sigma_{adm}$
($F > F_{adm}$)

$$\sigma_{adm} = 133,33 \text{ MPa}$$

$\Rightarrow$ Barra BH $\leadsto \sigma_{BH} = E \cdot \frac{\Delta L}{L_2} = \frac{E}{L_2} \cdot \frac{F}{A E \left(\frac{1}{L_2} + \frac{1}{L_1} + \frac{2 \sin \theta}{L_1} \right)}$

$$\therefore F = \sigma_{BH} L_2 A \left(\frac{1}{L_2} + \frac{1}{L_1} + \frac{2 \sin \theta}{L_1} \right)$$

$\uparrow$ Para evitar, $\sigma_{BH} = \sigma_{adm}$

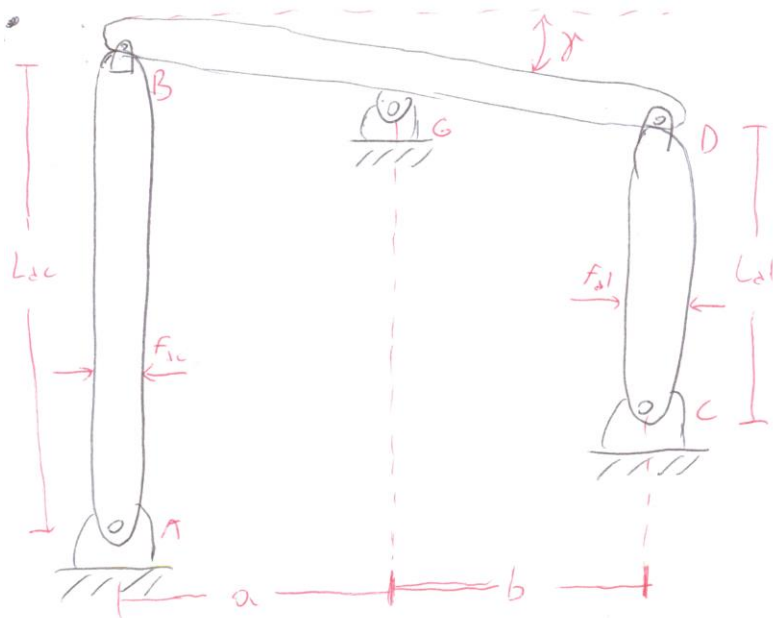
$$\Rightarrow F = 0,2524 \text{ MPa}$$

$\Rightarrow$ Barra BD $\leadsto F = \sigma_{BD} L_1 A \left(\frac{1}{L_2} + \frac{1}{L_1} + \frac{2 \sin \theta}{L_1} \right) = 0,5047 \text{ MPa}$

$\Rightarrow$ Barra B6 $\leadsto F = \sigma_{B6} \frac{L_1}{\sin \theta} A \left(\frac{1}{L_2} + \frac{1}{L_1} + \frac{2 \sin \theta}{L_1} \right) = 0,5828 \text{ MPa}$

* El valor operacional del sistema debe ser para el F más bajo. Por lo tanto

$$F_{max} = 0,2524 \text{ MPa} = 252,4 \text{ kPa}$$



* Deformación térmica

$$\epsilon = \alpha \cdot \Delta T$$

$$\frac{l_f - l_0}{l_0}$$

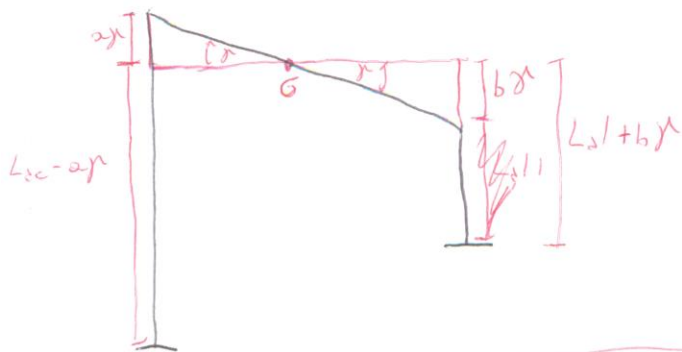
* Para que BGD quede horizontal, el acero debe acortarse y el aluminio alargarse, sin embargo ambas se alargan producto de $\Delta T \Rightarrow$ Utilizaremos ppio. de superposición

→ Asumimos que ambas barras se alargan térmicamente

→ Calculamos la def. elástica necesaria para que BGD quede horizontal

$$\left. \begin{aligned} \text{Por def. térmica: } L_{ac}^f &= L_{ac} (1 + \alpha_{ac} \Delta T) \\ L_{al}^f &= L_{al} (1 + \alpha_{al} \Delta T) \end{aligned} \right\} (*)$$

* Ahora suponemos que las barras se han alargado más de lo necesario, por lo que BGD es rígido, deben acortarse un poco:



• Estado inicial luego de def. térmica
(L_{ac}^f), (L_{al}^f)

• Estado final luego de def. elástica restitutoria
($L_{ac} - ax$), ($L_{al} + bx$)

$$\therefore \epsilon = \frac{l_f - l_0}{l_0} \Rightarrow$$

$$\epsilon_{ac} = \frac{L_{ac}^f - (L_{ac} - ax)}{L_{ac}^f} \quad \wedge \quad \epsilon_{al} = \frac{L_{al}^f - (L_{al} + bx)}{L_{al}^f}$$

• Aplicando Ley de Hooke

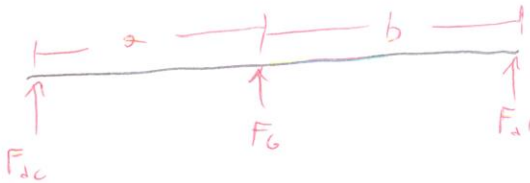
$$\frac{\sigma_{dc}}{E_{dc}} = \frac{F_{dc}}{A_{dc} E_{dc}} = \frac{L_{dc}^F - L_{dc} + \alpha \delta}{L_{dc}^F}$$

$$\frac{\sigma_{dl}}{E_{dl}} = \frac{F_{dl}}{A_{dl} E_{dl}} = \frac{L_{dl}^F - L_{dl} - b \delta}{L_{dl}^F}$$

• Aplicando ~~*~~ y despejando F_i :

$$\begin{aligned} F_{dc} &= A_{dc} E_{dc} \left(\frac{L_{dc}(1 + \alpha_{dc} \Delta T) - L_{dc} + \alpha \delta}{L_{dc}^F (1 + \alpha_{dc} \Delta T)} \right) \\ F_{dl} &= A_{dl} E_{dl} \left(\frac{L_{dl}(1 + \alpha_{dl} \Delta T) - L_{dl} - b \delta}{L_{dl}^F (1 + \alpha_{dl} \Delta T)} \right) \end{aligned} \quad (\star)$$

• ¿Qué falta? $\Rightarrow$ DCL en barra BGD



$$\sum_G M_z = 0 \Rightarrow F_{dc} \cdot a = F_{dl} \cdot b \quad (\dagger)$$

• Aplicando $(\star)$ en $(\dagger)$, obtendremos una expresión para ΔT :

$$A \Delta T^2 + B \Delta T + C = 0$$

$$\star A = \alpha_{dl} \alpha_{dc} L_{dc} L_{dl} (2 A_{dc} E_{dc} - b A_{dl} E_{dl})$$

$$\star B = [2 A_{dc} E_{dc} L_{dl} (L_{dc} \alpha_{dc} + \alpha \delta \alpha_{dl}) + b A_{dl} E_{dl} L_{dc} (L_{dl} \alpha_{dl} - b \delta \alpha_{dc})]$$

$$\star C = \delta [2^2 A_{dc} E_{dc} L_{dl} + b^2 A_{dl} E_{dl} L_{dc}]$$

• Resolviendo el sistema:

$$\Delta T = 31,2^\circ \checkmark \text{ ó } 32882^\circ \times \Rightarrow \boxed{\Delta T = 31,2^\circ}$$

b) Reemplazando el valor obtenido de ΔT en *

$$F_{d1} = 3197,55 \text{ N}$$

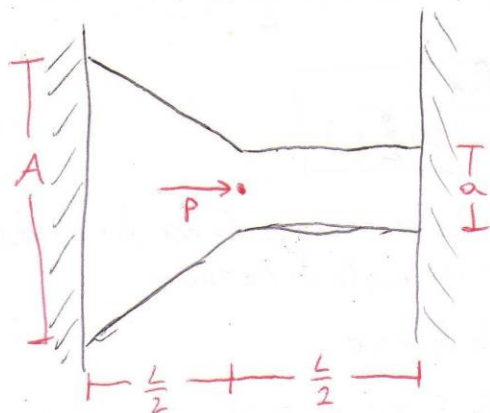
$$F_{dc} = 6395,11 \text{ N}$$

*Calculando las cargas críticas:

$$* P_{cr}^{dc} = \frac{\pi^2}{L_{dc}^2} E_{dc} \cdot \frac{f_{dc}^3 \cdot e_{dc}}{12} = 2229,37 \text{ N} \rightarrow \text{Falla por pandeo}$$

$$* P_{cr}^{d1} = \frac{\pi^2}{L_{d1}^2} E_{d1} \cdot \frac{f_{d1}^3 \cdot e_{d1}}{12} = 973,266 \text{ N} \rightarrow \text{Falla por pandeo}$$

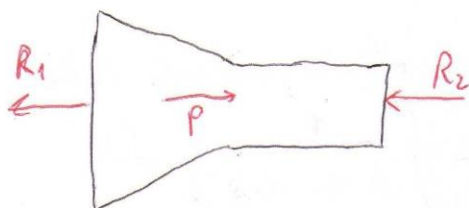
P3)



Espejor e
Módulo de Elasticidad E

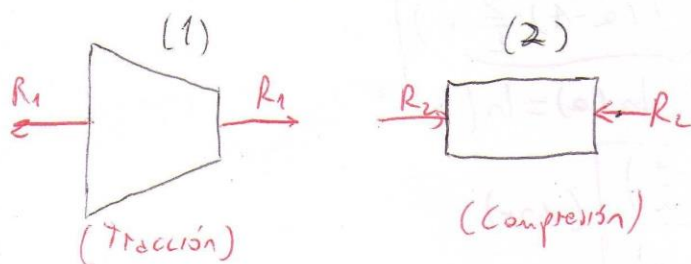
Principal problema: Área en el primer tramo varía con la distancia
 $\Rightarrow$ Esfuerzo varía con la distancia.

• DCL completo



$$\sum F_x = 0 \Rightarrow \boxed{P = R_1 + R_2} \text{ (i)}$$

• Cortes antes y después de P:



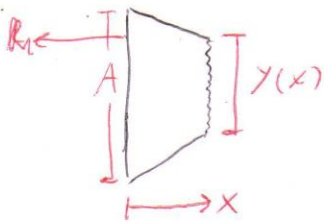
Como el largo total no cambia,
 lo que se alarga (1), se acorta (2)

$$\Rightarrow \boxed{\Delta L_1 = \Delta L_2} \text{ (ii)}$$

• Aplicación de Ley de Hooke:

$$(2): \sigma = \frac{R_2}{a \cdot e} \Rightarrow \frac{R_2}{a \cdot e} = E \cdot \frac{\Delta L_2}{L/2} \Rightarrow \boxed{\Delta L_2 = \frac{1}{2} \cdot \frac{R_2 L}{a \cdot e E}} \text{ (iii)}$$

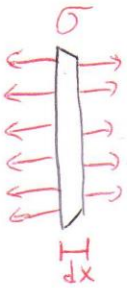
(17);



$$\sigma = \frac{R_1}{e \cdot y(x)} \quad \text{y} \quad y(x) = A + (a - A) \cdot \frac{2}{L} x$$

$$\sigma(x) = \frac{R_1}{e(A + (a - A) \frac{2}{L} x)}$$

* Cada segmento diferencial del tramo (1) se deforma en magnitudes diferentes ya que el esfuerzo cambia con x . Al hacer un corte diferencial:



$$\epsilon = \frac{\Delta L(dx)}{dx}$$

Lo que se alarga cada trozo diferencial
Longitudinal del trozo diferencial

Ley de Hooke

$$\sigma = E \cdot \epsilon \Rightarrow \epsilon = \frac{\sigma}{E}$$

$$\frac{\Delta L(dx)}{dx} = \frac{\sigma(x)}{E}$$

$$\Delta L(dx) = \frac{\sigma(x)}{E} dx$$

~~Integrando~~

* Integrando entre 0 y $\frac{L}{2}$, se obtiene el alargamiento total:

$$\int \Delta L(dx) = \Delta L_1 = \int_0^{\frac{L}{2}} \frac{\sigma(x)}{E} dx = \frac{R_1}{eE} \int_0^{\frac{L}{2}} \frac{1}{A + (a - A) \frac{2}{L} x} dx$$

$$\Rightarrow \Delta L_1 = \frac{R_1}{eE} \cdot \frac{L}{2(a - A)} \cdot \ln \left(A + (a - A) \cdot \frac{2}{L} x \right) \Big|_0^{\frac{L}{2}}$$

$\ln(A) - \ln(a) = \ln\left(\frac{A}{a}\right)$

$$\therefore \Delta L_1 = \frac{R_1 L \cdot \ln\left(\frac{A}{a}\right)}{2eE(a - A)} \quad (iv)$$

* Debido a (ii) $\Rightarrow$ (iii) = (iv):

$$\frac{1}{2} \frac{R_2 K}{aE} = \frac{R_1 K \cdot \ln\left(\frac{A}{a}\right)}{2eE(a - A)} \Rightarrow R_2 = \frac{a}{a - A} \ln\left(\frac{A}{a}\right) R_1 \quad (v)$$

* Reemplazando (v) en (i), se tiene finalmente que:

$$R_1 \left(1 + \frac{2}{a-A} \ln\left(\frac{A}{a}\right) \right) = P$$

$$\therefore R_1 = \frac{P}{1 + \frac{2}{a-A} \ln\left(\frac{A}{a}\right)}$$
$$R_2 = \frac{P}{1 + \frac{1}{\frac{2}{a-A} \ln\left(\frac{A}{a}\right)}}$$